

Robust boundary flow in chiral active fluidXiang Yang,^{1,2,*} Chenyang Ren^{1,*}, Kangjun Cheng,³ and H. P. Zhang^{1,4,‡}¹*School of Physics and Astronomy and Institute of Natural Sciences, Shanghai Jiao Tong University, Shanghai 200240, China*²*School of Physical Science and Technology, Soochow University, Suzhou 215006, China*³*Zhiyuan College, Shanghai Jiao Tong University, Shanghai 200240, China*⁴*Collaborative Innovation Center of Advanced Microstructures, Nanjing 210093, China*

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We perform experiments on an active chiral fluid system of self-spinning rotors in a confining boundary. Along the boundary, actively rotating rotors collectively drive a unidirectional material flow. We systematically vary rotor density and boundary shape; boundary flow robustly emerges under all conditions. Flow strength initially increases then decreases with rotor density (quantified by area fraction ϕ); peak strength appears around a density $\phi = 0.65$. Boundary curvature plays an important role: flow near a concave boundary is stronger than that near a flat or convex boundary in the same confinements. Our experimental results in all cases can be reproduced by a continuum theory with single free fitting parameter, which describes the frictional property of the boundary. Our results support the idea that boundary flow in active chiral fluid is topologically protected; such robust flow can be used to develop materials with novel functions.

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Active matter is composed of constituent units individually powered by internal or external energy sources. In the majority of current studies, local energy injection drives constituent unit's linear motion [1]. In these systems, a wide range of phenomena has been reported, including emergent collective motion [2,3], pattern formation [4–7], and phase segregation without attraction [8–10]. Besides linear motion, local energy injection can also cause constituent units to actively rotate. Biological examples of such chiral active matter include rotating bacteria [11,12], circling bacteria [13–15], and sperm cells [4,16] near surfaces, and magnetotactic bacteria in rotating fields [17,18]. Artificial chiral active systems have also been developed, such as colloids [19–25], millimeter-scale magnets [26,27], and rotating granular particles [28–31]. Multiple numerical and theoretical studies on chiral active fluid have been carried out [28,32–39].

Interacting active rotors can form a range of collective phenomena. One such phenomena is unidirectional material flow localized at rotor/solid [28,40], rotor/liquid [25], and rotor phase boundaries [29,34]. A continuum theory was developed to reproduce boundary flow in a driven granular system in a circular confinement [28]. Later, the same theory was compared with numerical data of confined rotors [40]. Recently, Dasbiswas, Mandadapu, and Vaikuntanathan [41] studied topological properties of the continuum theory; they showed that the emergence of the boundary flow in active chiral fluid can be understood as an example of topological protection at boundary [41] and that the boundary flow is

insensitive to boundary interactions and highly resistant to perturbations. Similar robustness has been extensively studied in many topologically nontrivial systems, such as mechanical lattice [42–44], electronic [45], and photonic [46] systems.

Here, we investigate boundary flow of individually driven, rotating particles in confining boundaries by experiment and theory. Our experiments show that boundary flow robustly emerges in all cases of various rotor densities and boundary shapes. To facilitate the comparison between experiments and theory, we use experimental observations to simplify the continuum theory [28] and carry out independent experiments to identify model parameters. Eventually, our experimental results in all cases can be reproduced by the continuum theory with single free fitting parameter, which describes the frictional property of the boundary.

II. EXPERIMENTAL METHODS AND RESULTS

Our rotor ($m = 15.3 \pm 0.2$ g in mass) is driven by two Hexbug robots. Each Hexbug, 4.3 cm long and 1.2 cm wide, houses a 1.5 V button cell battery that drives a vibration motor; we use fresh batteries in each new experiment and run experiments for less than 20 min to prevent battery power degrading. Hexbug body is supported by 12 flexible legs that all bend slightly backwards. When turned on, the vibration motor sets Hexbug into forward hopping motion on a solid (PMMA) substrate [47,48]. As shown in Fig. 1(a), two Hexbug robots in a rotor are glued to a foam disk (radius $a = 2.5$ cm) in opposite directions; they can generate a torque that spins the rotor with a spin rate about $\Omega_0 \approx 8.4$ rad/s, cf. Fig. 1(c). Rotors and the substrate are carefully balanced so that translational motion of an isolated rotor is suppressed, cf. Fig. 1(b). Our rotors respond linearly to external force and torque, as shown in Figs. 1(d), and 1(e); a detailed description of these experiments can be found in the Supplemental

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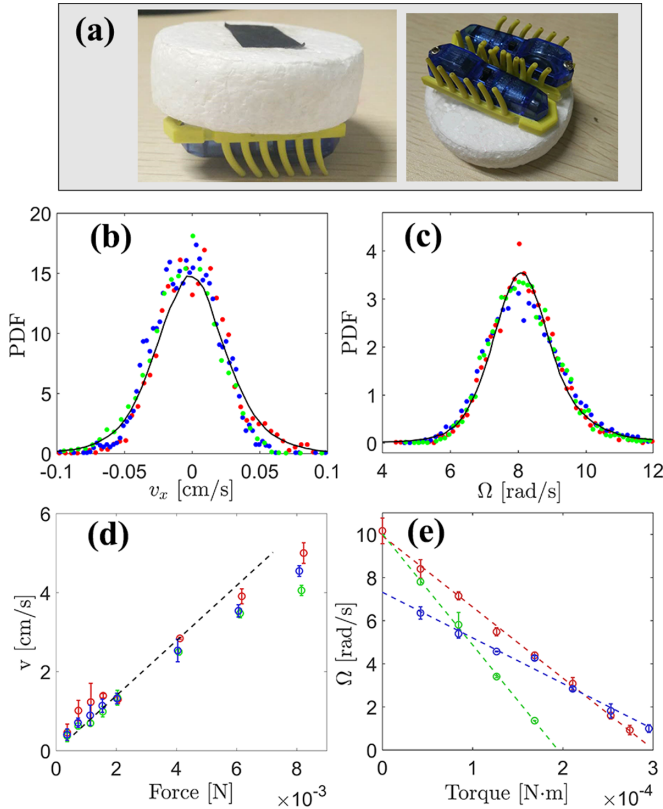


FIG. 1. Construction and properties of single rotor. (a) Two images showing a rotor made of two Hexbug robots and a foam disk. (b,c) Probability distribution functions of linear velocity and spin rate. (d,e) Rotor responses to external force and torque. Measurements from three representative rotors are shown as symbols in (b–e); data from the same rotor are shown by the same color. Lines in (b,c) represent average results over 40 rotors. Lines in (d,e) are linear fits to experimental data (symbols), and the error-bars represent standard deviations of measurements of a single rotor.

Material [49]. We note that rotors in our experiment have the same chirality, i.e., rotate in the same direction.

To observe localized boundary flow, we confine co-rotating rotors with solid boundaries which are precisely machined by a laser-cutter and covered with smooth tapes to reduce friction, cf. Fig. 2(a). Different numbers of rotors are used to vary density. Rotor motion is recorded by a digital camera at 30 frames per second; we use standard particle tracking method to measure rotor translation and rotation from recorded videos. Experimental results obtained in both axisymmetric and nonaxisymmetric confinements are discussed in detail below.

A. Axisymmetric boundary

We start from a circular boundary with a radius $R_c = 45$ cm, cf. Fig. 2(a). Five different numbers of rotors are used: $N = 160, 180, 190, 220,$ and 230 ; the corresponding area fractions $\phi \equiv \frac{N\pi a^2}{\pi R_c^2}$ are 0.49, 0.56, 0.59, 0.68, and 0.71, respectively. Typical system dynamics can be seen in [49]: while spinning rotors interact with neighbors and boundary, part of their angular momentum is converted to linear

momentum, which is reflected by rotors' translational motion; rotor translation is most pronounced near the boundary and is in the clockwise direction.

We measure spin rate Ω and velocity \vec{v} of each rotor and average measured results in 1.5×1.5 cm² bins. As shown in Figs. 2(b) and (e), coarse-grained spin rate $\Omega(\vec{r})$ has an approximately uniform spatial distribution and its mean value decreases as the rotor density increases, cf. Fig. 2(e); this is mainly caused by the frictional slides of neighbors. Coarsen-grained linear velocity field $\vec{v}(\vec{r})$ is shown in Fig. 2(c); local angular velocity $\zeta(\vec{r}) \equiv \frac{1}{2}(\nabla \times \vec{v}(\vec{r}))_z$ computed and plotted in Fig. 2(d). By averaging data in concentric annuli between $r - a$ and $r + a$, we can get radial profiles of $v_\theta(r)$ and $\zeta(r)$. Data in Figs. 2(f), 2(g) show that localized boundary flow emerges under all density conditions with different strength.

We add an inner boundary (15 cm in radius) to the system; this makes a ring-shaped confinement, as shown in Figs. 3(a). As in the case of circular boundary, coarsen-grained fields, $\vec{v}(\vec{r})$ and $\zeta(\vec{r})$, and their radial profiles are measured and plotted in Fig. 3. From these data, we see that, in addition to the clockwise flow along the outer boundary, a counterclockwise flow emerges near the inner boundary, which is weaker and can most clearly be seen from $v_\theta(r)$ profiles in Fig. 3(c).

B. Nonaxisymmetric boundary

We further investigate two cases of a nonaxisymmetric boundary: capsule-shaped and U-shaped confinements as shown in Figs. 4(a) and 4(d). In both cases, curvature changes along the boundary and affects the strength of boundary flow. To quantify this point, linear velocity and local angular velocity at representative points, square and circular symbols in (a) and (d), are computed at six rotor densities. Data in Figs. 4(b), and 4(c) show that boundary flow near concave boundary (red symbols) is stronger than that (blue symbols) near a flat region. In the case of U-shaped confinement, cf. Figs. 4(d)–4(f) and [49], concave boundary (red symbols) generates stronger flow than convex boundary (blue symbols). We also discover that the flow velocity peaks near the density $\phi = 0.65$ in both cases, as shown in Figs. 4(b) and 4(e).

III. THEORETICAL ANALYSIS OF EXPERIMENTAL RESULTS

A. Continuum theory

To understand experimental results in Figs. 1–4, we use a continuum theory developed by Tsai and coauthors [28]. The theory describes the conservation laws of the following hydrodynamic variables: the mass density $\rho(\vec{r}, t)$, the momentum density $\rho\vec{v}(\vec{r}, t)$, and the angular momentum density $I\Omega(\vec{r}, t)$, where I is the moment of inertial density. The first continuum equation describes mass conservation:

$$\frac{\partial \rho}{\partial t} + \nabla \cdot (\rho\vec{v}) = 0, \quad (1)$$

where the mass density ρ is proportional to the area fraction of rotors ϕ : $\rho = \frac{m}{\pi a^2}\phi$. Rotor density in our experiments is spatially homogeneous, cf. Fig. 2(a) and S2 in the Supplemental Material [49], which allows us simplify Eq. (1) as

$$\nabla \cdot \vec{v} = 0. \quad (2)$$

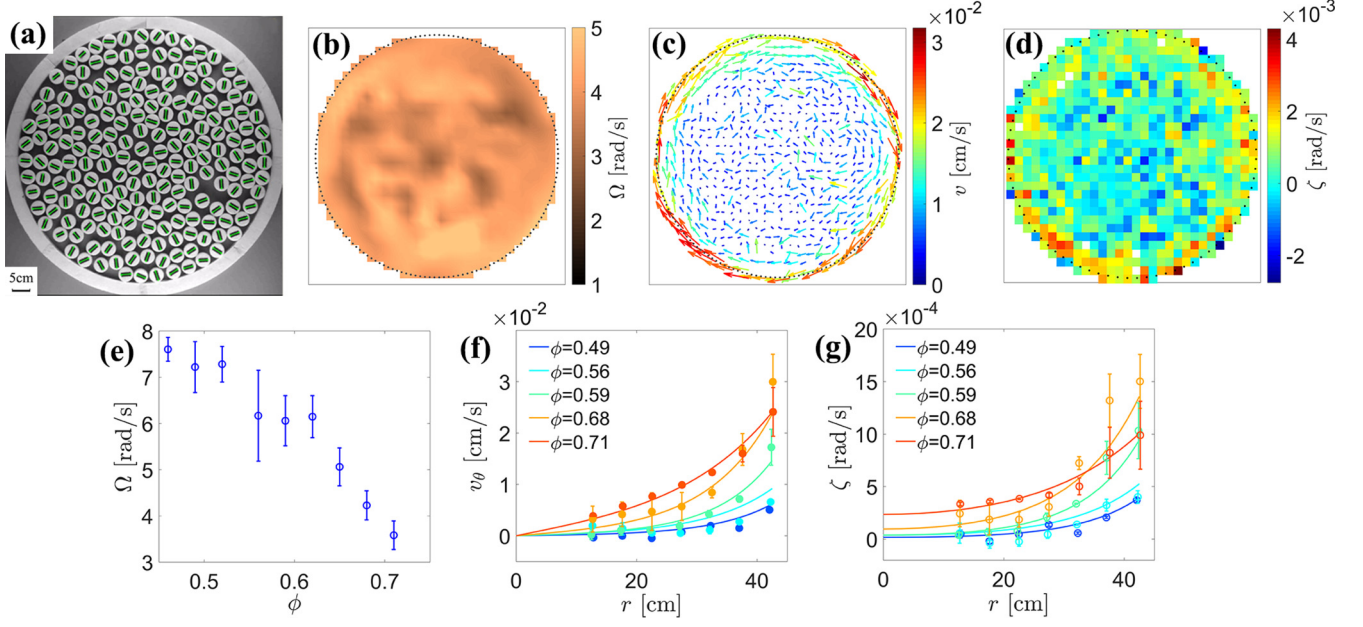


FIG. 2. Results in a circular confinement. (a) Snapshot from an experiment with $\phi = 0.68$. Black markers on rotors are used to facilitate particle tracking. Coarsen-grained fields [$\Omega(\vec{r})$, $\vec{v}(\vec{r})$ and $\zeta(\vec{r})$] from the same experiment are shown in (b–d); dotted line marks the effective boundary of the space accessible to the rotor center. (e) Density dependence of averaged spin rate Ω . (f,g) Radial profiles of $v_\theta(r)$ and $\zeta(\vec{r})$. symbols and lines represent experimental and numerical results, respectively. Error-bars in (e–g) represent the standard deviation of measured values.

The angular momentum of rotors is conserved:

$$ID_t\Omega = D_\Omega\nabla^2\Omega - \Gamma^\Omega\Omega - \Gamma(\Omega - \zeta) + \tau, \quad (3)$$

where $D_t \equiv \partial_t + \vec{v} \cdot \nabla$ is convective derivative, D_Ω is the angular momentum diffusion constant, Γ^Ω is the angular

friction coefficient due to rotor-substrate interaction, Γ is spin-velocity coupling constant, and τ stands for driving torque density field experienced by the rotors. We can simplify Eq. (3) as

$$\tau = \Gamma^\Omega\Omega + \Gamma\Omega$$

by the following experimental observations: (1) our system in steady state; (2) homogeneous angular momentum field Ω [Fig. 2(b)]; (3) local angular velocity ζ is much less than spin rate Ω [Figs. 2(g) and 3(d)]. Under low density condition, isolated rotors experiences little coupling to others, i.e., $\Gamma = 0$, we have spin rate for isolated rotors:

$$\Omega_0 = \frac{\tau}{\Gamma^\Omega}.$$

Combining two equations above, we have the following relation:

$$\frac{\Omega}{\Omega_0} = \frac{\Gamma^\Omega/\Gamma}{\Gamma^\Omega/\Gamma + 1}. \quad (4)$$

Momentum conservation requires

$$\rho D_t\vec{v} = -\nabla p + \eta\nabla^2\vec{v} - \Gamma^v\vec{v} + \frac{\Gamma}{2}\epsilon\nabla(\Omega - \zeta), \quad (5)$$

where η is the shear viscosity, Γ^v is the linear friction coefficient, and ϵ is two-dimensional antisymmetric symbol. The odd viscosity has been ignored in our system for quite large damping coefficient Γ^v [50]. With a steady-state assumption, we take the curl of Eq. (5)

$$((4\eta + \Gamma)\nabla^2 - 4\Gamma^v)\zeta - \Gamma\nabla^2\Omega = 0. \quad (6)$$

The above equation can be further simplified by assuming a homogeneous angular momentum field Ω and weak coupling

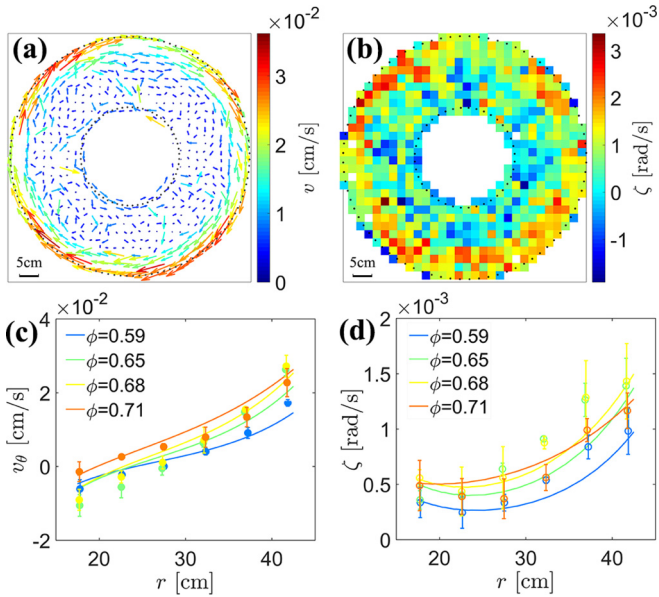


FIG. 3. Results in a ring-shaped confinement. (a) Velocity field and (b) local collective angular velocity field measured in experiments with $\phi = 0.68$. Radial profiles of $v_\theta(r)$ and $\zeta(\vec{r})$ are shown in (c,d); symbols and lines represent experimental and numerical results, respectively. Error-bars in (c,d) represent standard deviation of measured values.

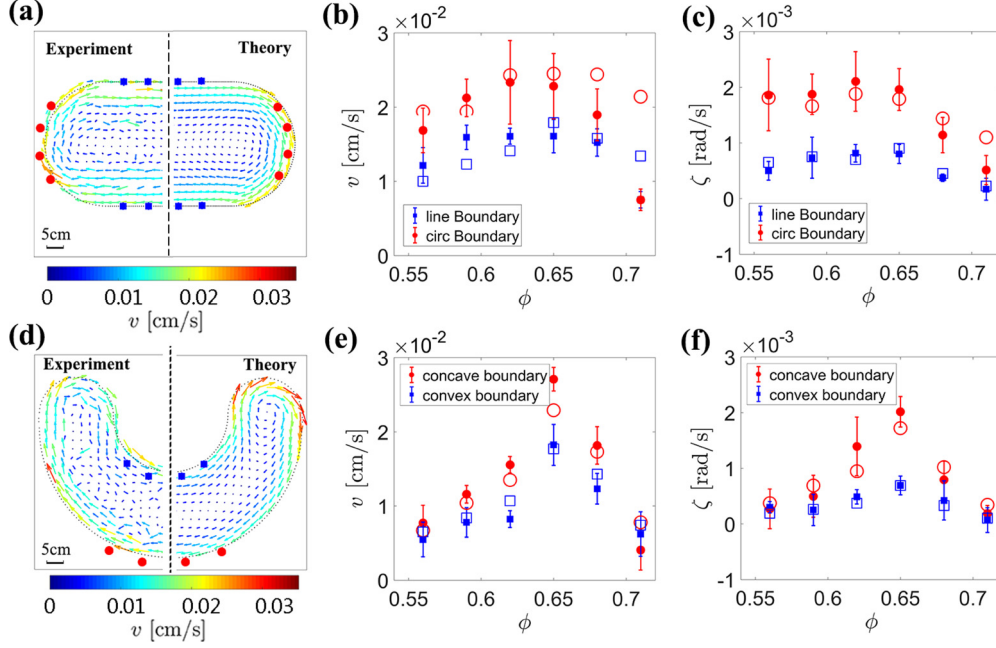


FIG. 4. Results in two nonaxisymmetric cases: (a–c) capsule-shaped and (d–f) U-shaped confinements. Effective boundary of rotor centers in two cases is represented by dotted line in (a) and (d). Experimentally measured velocity fields with $\phi = 0.68$ are shown in the left half of (a) and (d) with corresponding theoretical prediction on the right. Averaged velocity and local angular velocity at representative points, marked by symbols in (a) and (d), are computed at different densities and plotted in (b,c) and (e,f); experimental and theoretical results are represented by filled and empty symbols, respectively. Error-bars in (b,c,e,f) represent standard deviations of measured values.

$\Gamma \ll \eta$ [see Fig. 5(b)]; we end up with the following equation:

$$\left(\nabla^2 - \frac{\Gamma^v}{\eta}\right)(\nabla \times \vec{v})_z = 0. \quad (7)$$

B. Boundary conditions

Equations (2) and (7) can be solved with proper boundary conditions. The boundary is characterized by a local outward normal vector, \hat{r} , and a tangential direction, $\hat{\theta}$; the local radius curvature is denoted as R with the convention that a concave boundary has a positive radius of curvature. A rigid wall requires the radial velocity component to be zero:

$$v_{r,B} = 0, \quad (8)$$

where the subscript “ B ” stands for the boundary of the occupied region for rotor centers, as shown by the dotted line in

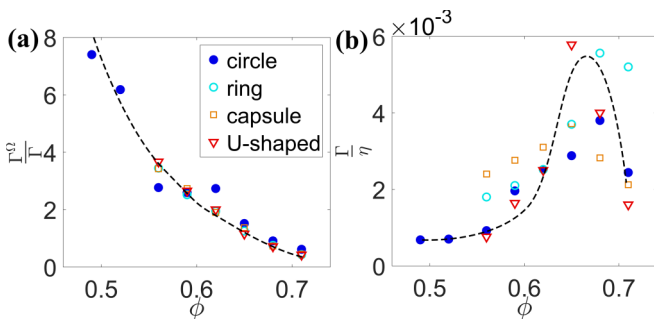


FIG. 5. Density dependence of two parameter ratios extracted from experiments: (a) $\frac{\Gamma^\Omega}{\Gamma^v}$ and (b) $\frac{\Gamma^\Omega}{\eta}$. Data from different confinements are shown by symbols. Dashed lines are a guide to the eye.

Figs. 2(b)–2(d) for circular boundary. The second boundary condition arises from the fact that rotors also experience a frictional force from the boundary; this leads to a tangential-radial component of the stress tensor

$$\sigma_{\theta r,B} = f_B, \quad (9)$$

where f_B is an effective boundary friction on unit length. We assume that f_B is proportional to the shear stress from the spin-velocity coupling

$$f_B = -k\Gamma\Omega. \quad (10)$$

As the friction of rotor-rotor and rotor-boundary have similar dependence on ϕ , the proportion factor k in Eq. (10) is treated as a constant in a given experiment. We express $\sigma_{\theta r}$ in velocity components, combine Eqs. (8)–(10) and obtain the following boundary condition (detailed discussions in [49]):

$$\left(\zeta - \frac{v_\theta}{R}\right)_B = \frac{\Gamma}{4\eta}(1 - 2k)\Omega_B. \quad (11)$$

C. Determination of model parameters

Figure 1(d) and (e) shows linear responses of isolated rotors to external force and torque. By measuring slopes of data in these plots, we extracted linear and angular frictional coefficient for isolated rotors: $\gamma^v = 0.14$ kg/s and $\gamma^\Omega = 0.32$ kg cm²/s. These two quantities are related to Γ^v and Γ^Ω as $\Gamma^v = \rho\gamma^v = \frac{m}{\pi a^2}\phi\gamma^v$ and $\Gamma^\Omega = \rho\gamma^\Omega = \frac{m}{\pi a^2}\phi\gamma^\Omega$; this leads to $\Gamma^\Omega/\Gamma^v = 0.44$ cm⁻² for all densities.

Equation (4) relates spin rate Ω to the ratio of angular frictional coefficient Γ^Ω to coupling constant Γ . Our experiments show that spin rate Ω decreases with increasing rotor density, cf. Fig. 2(e). From such data, we can use Eq. (4)

to measure the ratio Γ^Ω/Γ in different confinements and different densities. Results are plotted in Fig. 5(a), showing a monotonic decrease with area fraction ϕ ; collapse of all data on a single curve demonstrates that this ratio depends weakly on boundary shape and is a bulk property of the system.

We can estimate the ratio Γ/η from stress boundary condition, Eq. (11), by rewriting the equation as

$$\frac{\Gamma}{\eta} = \frac{4(\zeta - v_\theta/R)_B}{(1 - 2k)\Omega_B}.$$

Quantities in the above equation, ζ , v_θ , and Ω at boundary, can be measured directly from experiments. Therefore, for any given proportion constant k , we can compute $\frac{\Gamma}{\eta}$ along the boundary then average computed values, which depend weakly on local curvature. Averaged results for $\frac{\Gamma}{\eta}$ obtained with $k = 0.4$ are plotted in Fig. 5(b). $\frac{\Gamma}{\eta}$ results from different confinements approximately collapse onto a single curve and show a peak around density $\phi = 0.65$, where peak boundary flow in Figs. 4(b) and (e) appears. Figure 5(b) shows that spin-velocity coupling is weak in our system, with a coupling constant Γ two-order magnitude smaller than the shear viscosity η .

D. Comparison between theoretical and experimental results

With the process in the previous section, we can estimate three parameter ratios in every experiment. From these ratios, the only parameter in Eq. (7) can be determined:

$$\frac{\Gamma^v}{\eta} = \left(\frac{\Gamma^v}{\Gamma^\Omega}\right) \left(\frac{\Gamma^\Omega}{\Gamma}\right) \left(\frac{\Gamma}{\eta}\right). \quad (12)$$

Because spin rate Ω is spatially homogeneous, we set its boundary value in Eq. (11) Ω_B as the measured spin rate in bulk. We can determine all independent parameters except k through experimental measurements. For a given k value, we use a finite-element package (COMSOL) to solve Eqs. (2) and (7) with boundary conditions Eqs. (8) and (11) for steady flow $\vec{v}(\vec{r}, t)$ in all experiments. In a typical calculation, more than 10^4 finite elements are used to ensure convergence. Numerical results for a capsule-shaped confinement can be found in Fig. S4 of the Supplemental Material [49].

We theoretically compute radial velocity profiles $v_\theta(r)$ in circular confinement [c.f. Fig. 2(f)] for a range of k values and find that $k = 0.4$ leads to best fits between theory and experiments. We then fix $k = 0.4$ and use the same value for all other boundaries in Figs. 2–4. In the case of circular boundary, theoretical solutions correctly capture the spatial length scale and density dependence of the boundary flow, as shown in Fig. 2. In ring geometry, Fig. 3, continuum theory predicts the reversal of flow direction as one moves from the inner to outer boundary and nonmonotonic behavior in local angular velocity, ζ . In nonaxisymmetric cases, main features of steady flow are well captured in theoretical solutions, especially how flow depends on local curvature and rotor density. Effect of local curvature is manifested through the stress boundary condition [49], Eq. (11). Rotor density enters the theory through model parameters shown in Fig. 5.

We note that Figs. 4(b) and 4(e) shows some deviation of theoretical results from experiments at high densities. This is likely caused by transient jamming of densely packed rotors in experiments, which are not captured in the current fluid-based continuum theory. Transient jamming and associated elastic stress can also explain the sharp drop of boundary flow beyond density $\phi = 0.65$, cf. Figs. 4(b) and 4(e).

IV. CONCLUSION AND DISCUSSION

We have studied collective dynamics of rotors in various confining boundaries and density conditions. Actively rotating rotors collectively drives a unidirectional material flow along boundary. Boundary flow robustly emerges in all experiments with different rotor densities and boundary shapes. We showed that flow strength initially increases then decreases with rotor density and peak strength appears around a density $\phi = 0.65$. Boundary curvature plays an important role: flow near a concave boundary (with a positive radius of curvature) is stronger than that near a flat or convex boundary in the same confinements. We corroborate experimental measurements with a theoretical analysis based on a continuum theory, which is simplified under our experimental conditions; independent experimental measurements were used to determine transport coefficients in the theory. We demonstrated that our experimental results in all cases were quantitatively reproduced by the theory with single free fitting parameter $k = 0.4$, which describes the frictional property of the boundary. We have used foam boundary without smooth tape; boundary flow becomes very weak due to the increase of rotor-boundary friction. We have also explored the role of k parameter via hydrodynamic equations. Results are shown in supporting Fig. S5 of the Supplemental Material [49]: in circular confinement, boundary flow diminishes at $k = 0.5$ and reverses direction for $k > 0.5$.

Our experiments have shown that the boundary flow of rotors is unidirectional and insensitive to continuous changes of the boundary shape; these features are commonly observed in classical chiral systems with topological protections [51–53]. Furthermore, the hydrodynamic theory we used has been mapped to the classical Su-Schrieffer-Heeger model [41]; such a mapping allows us to interpret the observed unidirectionality and robustness against boundary shape change from a topological protection point of view. Topological nature of the boundary flow may allow us to develop new materials with novel and robust functions.

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